

University of California at Berkeley
College of Engineering
Department of Mechanical Engineering

ME102B, Fall 2014

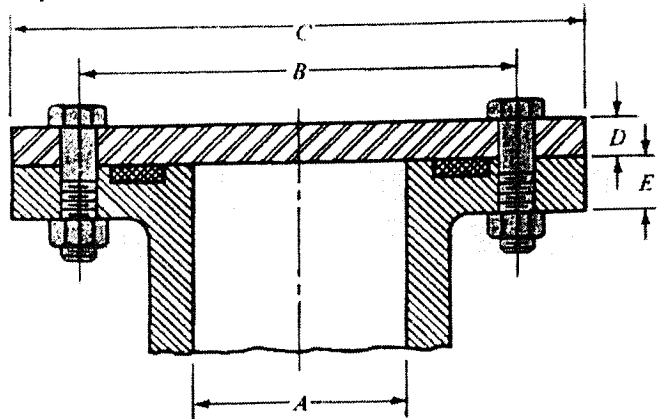
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Exam (November 12, 2014)
Close book, Open one formula sheet

Problem 1 (40%)

The cover of a pressure vessel is held in place by ten 1/2" UNC bolts (proof stress = 85 kpsi). The pressure is 200 Psi and effective area of the cover exposed to the pressure is 314 in². The ratio of stiffness of the bolt to the connected member is 1/3. Each bolt is tightened to 7500 lb initially, before the pressure is applied.

- (a) Draw a diagram to illustrate safety factors of bolts by showing the important lines in the figure. Please shade the area that is safe to operate and show the definition of safety factor against yielding and safety factor against separation. Derive the equations for safety factors against yielding and against separation, respectively (10%)
- (b) Calculate the safety factor against yielding. (5%)
- (c) Calculate the safety factor against separation. (5%)
- (d) Under what pressure, "separation" will start to occur? (5%)
- (e) Explain in less than 20 words what is happening after the separation? (5%)
- (f) Please calculate the safety factor against yielding (proof stress) right before and right after separation. (10%)



Problem 2 (60%)

The 24-tooth, 2mm module, 20° pinion shown as element 2 in the figure below rotates clockwise at 1000 rpm and is driven at a power of 2000 Watts. Gears 4, 5, and 6 have 24, 36, and 144 teeth, respectively and the gear efficiency is 100 percent.

- (a) Illustrate and explain the “contact ratio” and “interference” in a figure when two gears are meshing together. (5%)
- (b) What is the contact ratio for gear 4 and 5? (10%)
- (c) Determine if there is interference between gear 4 and 5? (5%)
- (d) What is the rotational speed and direction of the arm 3? (10%)
- (e) What torque can arm 3 deliver? (10%)
- (f) Draw free-body diagrams of the arm 3 and Gears 2, 4, 5 respectively, and show/analyze/calculate all forces which act upon them in both figures. (20%)

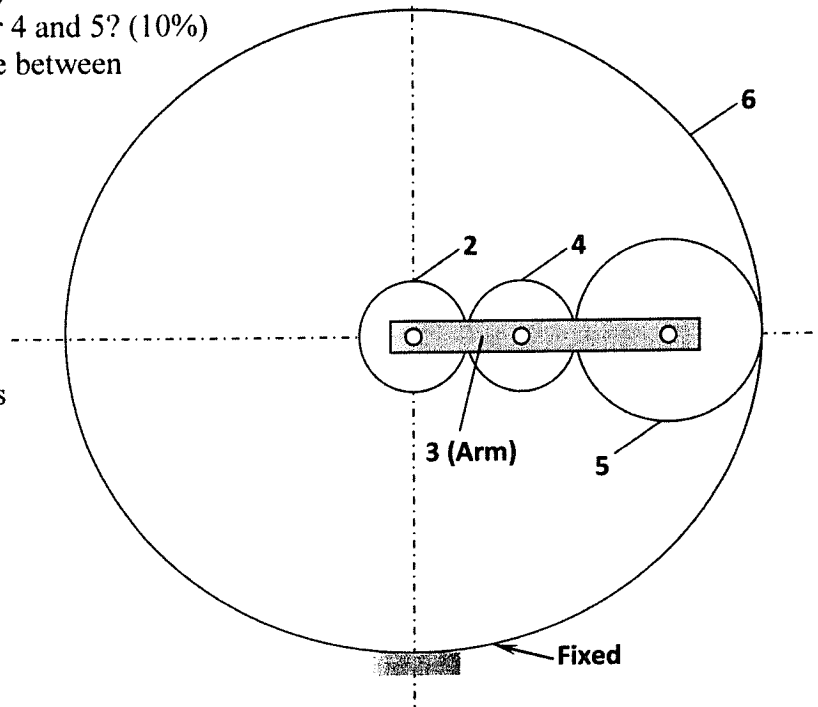


Table 8-2
Diameters and Area of Unified Screw Threads UNC and UNF*

Nominal Diameter in	Coarse Series - UNC			Fine Series - UNF		
	Threads per Inch N	Tensile Stress Area A_t in ²	Minor Diameter Area A_m in ²	Threads per Inch N	Tensile Stress Area A_t in ²	Minor Diameter Area A_m in ²
0	0.0600					
1	0.0730					
2	0.0860	64	0.00263	80	0.00180	0.00151
3	0.0990	56	0.00370	72	0.00278	0.00237
4	0.1120	48	0.00487	64	0.00394	0.00339
5	0.1250	40	0.00604	56	0.00523	0.00451
6	0.1380	40	0.00796	48	0.00661	0.00566
8	0.1640	32	0.00909	44	0.00880	0.00716
10	0.1900	32	0.0140	40	0.01015	0.00874
12	0.2160	24	0.0175	36	0.01474	0.01285
14	0.2500	24	0.0242	32	0.0200	0.0175
16	0.3125	20	0.0318	28	0.0258	0.0226
18	0.3750	18	0.0324	28	0.0364	0.0326
20	0.4375	16	0.0775	24	0.0580	0.0524
22	0.5000	14	0.1063	24	0.0878	0.0809
24	0.5625	14	0.1419	20	0.1187	0.1090
27	0.6250	13	0.182	20	0.1599	0.1486
30	0.7500	12	0.226	18	0.203	0.189
36	0.8750	11	0.334	18	0.256	0.240
42	1.0000	10	0.462	16	0.373	0.351
48	1.1250	9	0.606	14	0.509	0.480
54	1.2500	8	0.969	12	0.663	0.625
60	1.5000	7	1.405	12	1.073	1.024
66		6	1.294	12	1.581	1.521

*The table was compiled from ANSI B1.1-1974. The minor diameter was found from the equation $d_m = d - 1.299(0.88)p$, and the pitch diameter from $d_p = d - 0.649(515)p$. The mean of the pitch diameter and the minor diameter was used to compute the tensile stress area.

Figure 8-3
 (a) Square thread; (b) Acme thread

